

WAVES

REVIEW.

NORMAL MODES

$$[m^{-1}K - \omega^2 I]A = 0$$

Find ω^2 :

$$\det [m^{-1}K - \omega^2 I] = 0$$

K MATRIX: $k_{ij} = - \frac{F_{xj}}{x_j}$

~~#~~ FORCE ON i BY j

① HOLD i , MOVE j BY x_j

② DETERMINE FORCE ON i

③ DIVIDE BY x_j AND MULTIPLY BY -1 .

M IS DIAGONAL w/ MASSES ALONG
DIAGONAL. WE DEAL ONLY w/ EQUAL MASSES

INVERSE OF A MATRIX B

$$B^{-1} = \frac{B^T}{|B|} \leftarrow \text{DETERMINANT}$$

WAVES REVIEW. (NORMAL WAVES)

$$(B_c)_{ij} = (-1)^{i+j} \left[\text{DETERMINANT OF} \right. \\ \left. \text{MATRIX w/ row } i \text{ AND} \right. \\ \left. \text{row } j \text{ DELETED} \right]$$

$\Rightarrow T$ MEANS TRANSPOSE (ROW #'S
GET SWAPPED w/ COLUMN #'S)

MOTION OF SYSTEM

$$X(t) = \sum_{d=1}^n b_d A^d \cos \omega_d t + c_d A^d \sin(\omega_d t)$$

REAL

$$= \sum_{d=1}^n d_d A^d \cos(\omega_d t - \theta_d)$$

REAL

b_d & c_d (or d_d & θ_d) DETERMINED
FROM INITIAL CONDITIONS.

WAVES.

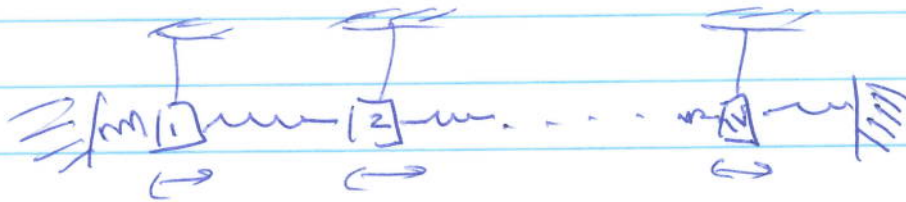
SYMMETRIES.

SOLVE EIGENVALUE PROBLEM

$$S A^m = \beta_m A^m$$

SEE EXAMPLE IN TEXT

SPACE TRANSLATION. INVARIANCE.



NORMAL MODES

$$A_j^m = \sin \frac{j m \pi}{N+1}, \quad m=1, 2, \dots, N$$

$$j=1, 2, \dots, N$$

m LABELS NORMAL MODES

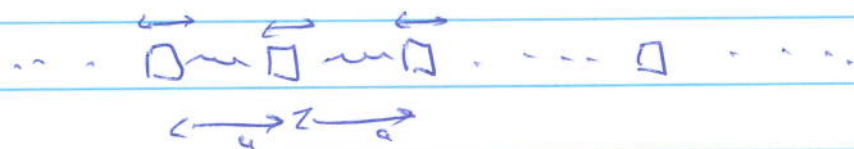
j LABELS BLOCKS,

EIGEN FREQUENCIES: :

$$\omega^2 = z_B - z_C \cos \left(\frac{m \pi}{N+1} \right)$$

$$\omega / z_B = m g / l + z k \quad C = k$$

WAVES. REVIEW.



Normal modes for infinite system.

$$A(x) = e^{ikx} \quad \text{w/} \quad -\frac{\pi}{a} < \text{Re}k \leq \frac{\pi}{a}$$

$$A(x+\lambda) = A(x) \quad \text{w/} \quad \lambda = 2\pi/k$$

Finite

$$A^n(x) = \sin kx$$

$$\text{w/} \quad k = \frac{n\pi}{L}$$

$$L = (n+1)a$$

$$n = 1, 2, \dots, N$$

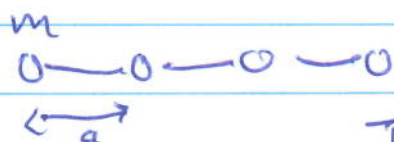
normal modes

DISPERSION RELATION

EQN THAT RELATES WAVE NUMBER k AND ANGULAR FREQUENCY ω .

EXAMPLE: INFINITE BEADED STRING

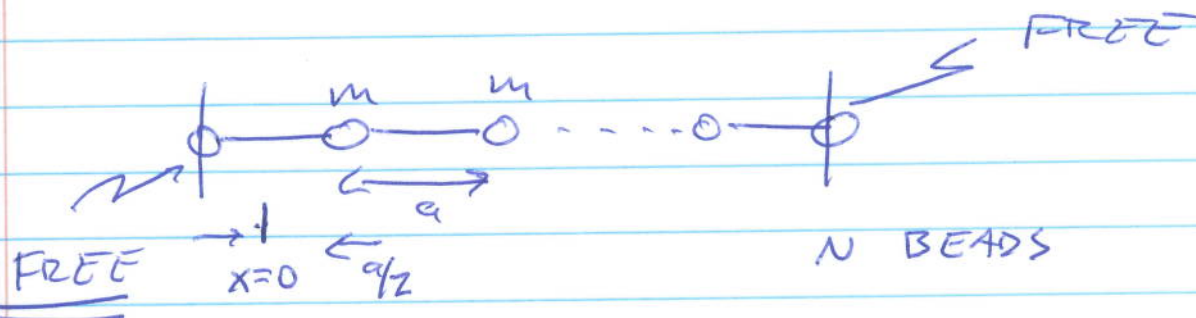
$$\omega^2 = \frac{4T}{ma} \sin^2 ka/2$$



T = STRING TENSION.

WAVE REVIEW

BEADED STRING

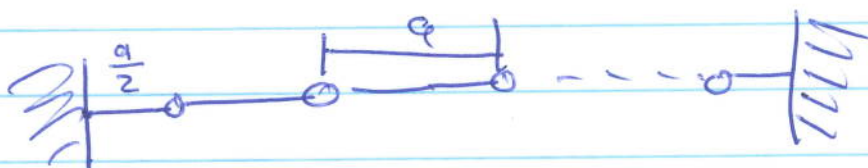


$$A_j = \cos \left[ka \left(j - \frac{1}{2} \right) \right], \quad k = \frac{n\pi}{Na}$$

$$\omega^2 = \frac{4T}{ma} \sin^2 \frac{ka}{2} \quad \begin{array}{l} n = 0, 1, \dots, N-1 \\ j = 1, \dots, N \end{array}$$

A_j DESCRIBES TRANSVERSE DISPLACEMENT FROM EQUILIBRIUM.

FIXED ENDER



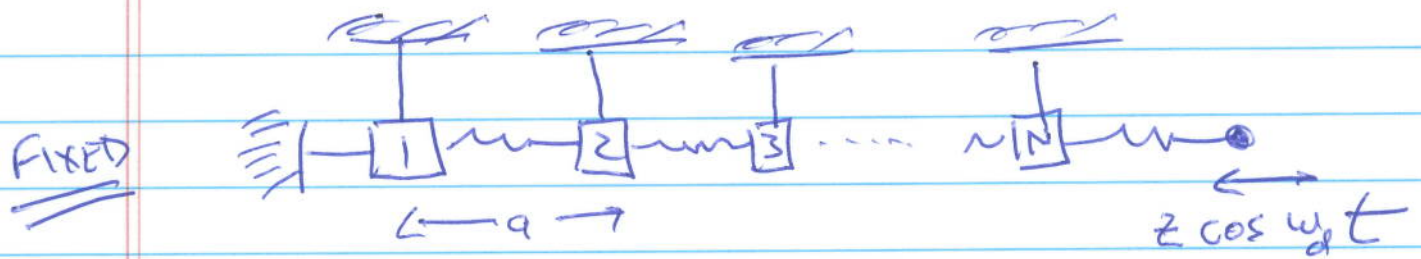
$$A_j = \sin \frac{n\pi \left(j - \frac{1}{2} \right)}{Na} = \sin \left[ka \left(j - \frac{1}{2} \right) \right]$$

$$\omega^2 = \frac{4T}{ma} \sin^2 \frac{ka}{2}$$

$$\begin{array}{l} n = 0, 1, 2, \dots, N-1 \\ j = 1, 2, \dots, N \end{array}$$

WAVE REVIEW.

DRIVEN / FORCED OSCILLATORS



$$\psi(x,t) = y \sin kx e^{-i \omega t}$$

$$\omega / y = \frac{z}{\sin kL}$$

$$L = (N+1)a$$

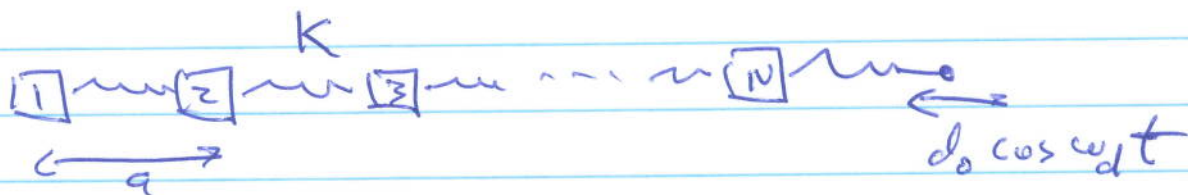
$$k = \frac{1}{a} \cos^{-1} \frac{zB - \omega^2 d}{zC}$$

$\omega / B \& C:$ $zB = \frac{mg}{l} + 2k$ (spring)

$C = k$ (spring const.)

WAVE REVIEW

FREE END FORCED OSCILLATION.



$$\psi_s(t) = \frac{\cos\left[\frac{(2s-1)ka}{2}\right]}{\cos\left[\frac{(2N+1)ka}{2}\right]}$$

s = block label
N = # BLOCKS

$$\omega_d^2 = \frac{4K}{m} \sin^2 \frac{ka}{2}$$

WAVE REVIEW

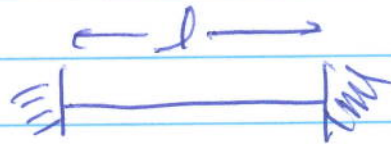
CONTINUOUS STRINGS

WAVE EQU:

$$\frac{\partial^2 \psi(x,t)}{\partial t^2} = \frac{T}{\rho L} \frac{\partial^2 \psi(x,t)}{\partial x^2}$$

w/ DISPERSION RELATION $\omega^2 = \frac{T}{\rho L} k^2$

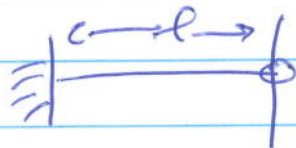
FIXED ENDS



$$\psi(x) = \sum_{n=1}^{\infty} c_n \sin \frac{n\pi x}{l}$$

$$c_n = \frac{2}{l} \int_0^l dx \sin \frac{n\pi x}{l} \psi(x)$$

1-FREE END



$$\psi(x) = \sum_{n=1}^{\infty} c_n \sin \frac{(2n-1)\pi x}{2l}$$

$$c_n = \frac{2}{l} \int_0^l dx \sin \frac{(2n-1)\pi x}{2l} \psi(x)$$